

Simulation and Validation of a Three-Phase Flow Model

CONTEXT

The modeling and the accurate simulation of *steam explosion* is an important topic for nuclear safety analysis. This phenomenon may occur when some very hot liquid (molten metal for instance) flows down in a quiet liquid component such as water. In this case the heat transfer might lead to the rapid creation of large volumes of water vapour, so rapid that it can be *explosive* [1].

THREE-PHASE FLOW MODEL

Within each phase, $\alpha_k \in [0,1]$, ρ_k , $m_k = \alpha_k \rho_k$, U_k , p_k ,

• Step 2 - Relaxation step : Starting with W^{n+1,-}, compute Wⁿ⁺¹ approximate solution of:

$$\begin{cases} \partial_t \alpha_k = S_k^{\alpha}(\mathbf{W}) \\ \partial_t m_k = 0 \\ \partial_t (m_k \mathbf{U}_k) = \mathbf{S}_k^U(\mathbf{W}) \\ \partial_t (\alpha_k E_k) - \sum_{l=1, l \neq k}^3 \prod_{kl} (\mathbf{W}) \frac{\partial \alpha_l}{\partial t} = S_k^E(\mathbf{W}) \end{cases}$$

This time scheme complies with the entropy inequality (5). A 3D Finite Vol-

 $e_k(p_k, \rho_k)$ and $E_k = 1/2\rho_k \mathbf{U}_k \cdot \mathbf{U}_k + \rho_k e_k(p_k, \rho_k)$ represent respectively the statistical fraction, the mean density, the partial mass, the mean velocity, the mean pressure, the mean internal energy and the mean total energy of phas $k = 1 \rightarrow 3$, we note $\mathbf{W} = (\alpha_2, \alpha_3, m_1, m_2, m_3, m_1 \mathbf{U}_1, m_2 \mathbf{U}_2, m_3 \mathbf{U}_3, \alpha_1 E_1, \alpha_2 E_2, \alpha_3 E_3)^t$ the state variable. The set of PDEs that is considered is (see [6]):

$$\begin{cases} \frac{\partial \alpha_{k}}{\partial t} + \mathbf{V}_{i}(\mathbf{W}) \cdot \nabla \alpha_{k} = S_{k}^{\alpha}(\mathbf{W}) \\ \frac{\partial m_{k}}{\partial t} + \nabla \cdot (m_{k}\mathbf{U}_{k}) = 0 \\ \frac{\partial m_{k}\mathbf{U}_{k}}{\partial t} + \nabla \cdot (m_{k}\mathbf{U}_{k} \otimes \mathbf{U}_{k} + \alpha_{k}p_{k}\mathbf{Id}) + \sum_{l=1, l \neq k}^{3} \Pi_{kl}(\mathbf{W}) \nabla \alpha_{l} = \mathbf{S}_{k}^{U}(\mathbf{W}) \\ \frac{\partial \alpha_{k}E_{k}}{\partial t} + \nabla \cdot (\alpha_{k}E_{k}\mathbf{U}_{k} + \alpha_{k}p_{k}\mathbf{U}_{k}) - \sum_{l=1, l \neq k}^{3} \Pi_{kl}(\mathbf{W}) \frac{\partial \alpha_{l}}{\partial t} = S_{k}^{E}(\mathbf{W}) \end{cases}$$
(1)

The statistical fraction α_1 complies with: $\alpha_1 = 1 - \alpha_2 - \alpha_3$, since phases are immiscible. We restrict here to the case where: $\mathcal{V}_i(\mathbf{W}) = \mathbf{U}_1$ (phase 1 denotes the liquid metal), thus following [6]:

$$\Pi_{12}(\mathbf{W}) = \Pi_{21}(\mathbf{W}) = \Pi_{23}(\mathbf{W}) = p_2$$

$$\Pi_{13}(\mathbf{W}) = \Pi_{31}(\mathbf{W}) = \Pi_{32}(\mathbf{W}) = p_3$$
(2)

Closure laws for $S_k^{\alpha}(\mathbf{W}), \mathbf{S}_k^U(\mathbf{W}), S_k^E(\mathbf{W})$ take the form:

$$\begin{cases} S_k^{\alpha}(\mathbf{W}) = \sum_{l=1, l \neq k}^3 \frac{\alpha_k \alpha_l}{\tau_{kl}(\mathbf{W})} (p_k - p_l) \\ \mathbf{S}_k^U(\mathbf{W}) = \sum_{l=1, l \neq k}^3 \mathbf{D}_{kl}(\mathbf{W}) \\ S_k^E(\mathbf{W}) = \sum_{l=1, l \neq k}^3 \mathbf{V}_{kl}(\mathbf{W}) \cdot \mathbf{D}_{kl}(\mathbf{W}) + \sum_{l=1, l \neq k}^3 \psi_{kl} \end{cases}$$
(3)

ume scheme is built. The first evolution step - that accounts for convective effects - involves an explicit scheme (Rusanov scheme was implemented), whereas the relaxation step is implicit.

NUMERICAL RESULTS

Verification test case : Riemann problem

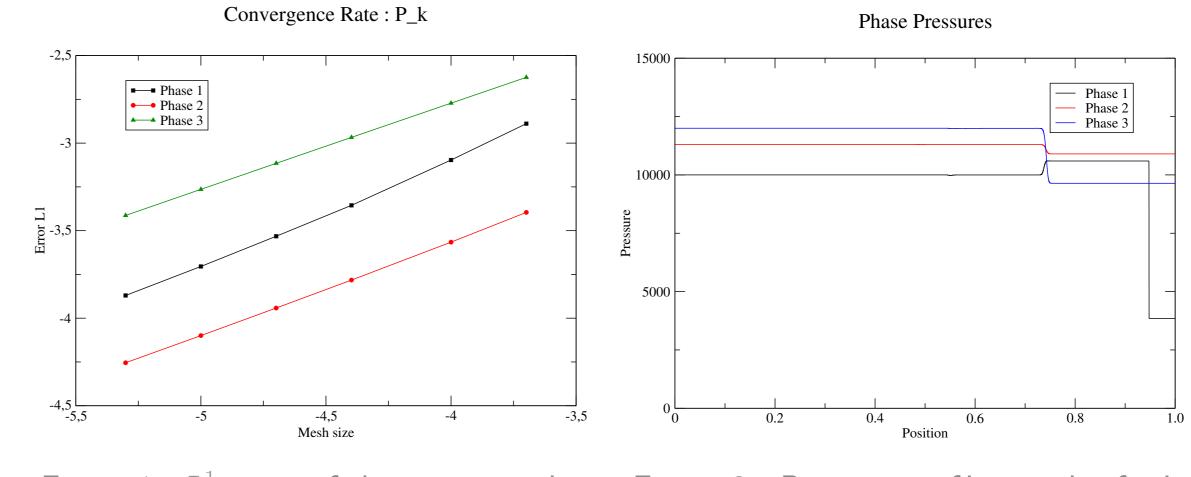


Figure 1: L^1 norm of the error wrt the mesh size h Figure 2: Pressure profiles at the final time T = 8ms.

Validation test case :

See [3] for the experimental setup, and [2] for a barotropic approach:

2.7	3.0	3.2	3.7	
x	x	x	x	

where $\tau_{kl}(\mathbf{W}) = \tau_{lk}(\mathbf{W})$, $\mathbf{D}_{kl}(\mathbf{W}) = e_{kl}(\mathbf{W})(\mathbf{U}_l - \mathbf{U}_k)$ with $e_{kl}(\mathbf{W}) = e_{lk}(\mathbf{W})$, and $\mathbf{V}_{kl} = \frac{1}{2}(\mathbf{U}_k + \mathbf{U}_l)$. τ_{kl} and e_{kl} are positive bounded functions (see [4] for pressure relaxation time scales). The heat transfer term is given by $\psi_{kl} = \frac{a_k - a_l}{\tau_{kl}^{\psi}}$, where $\tau_{kl}^{\psi} = \tau_{lk}^{\psi}$ is the characteristic time of heat transfer term is given between phases k and l, and $a_k = (s_k)^{-1} \left(\frac{\partial s_k(p_k,\rho_k)}{\partial p_k}\right) \left(\frac{\partial e_k(p_k,\rho_k)}{\partial p_k}\right)^{-1}$. We define $s_k(p_k,\rho_k)$, the specific entropy of phase k: $c_k^2 \frac{\partial s_k(p_k,\rho_k)}{\partial p_k} + \frac{\partial s_k(p_k,\rho_k)}{\partial \rho_k} = 0$, and also the mixture entropy $\eta(\mathbf{W})$ and the entropy flux $\mathbf{F}_{\eta}(\mathbf{W})$:

$$\begin{cases} \eta(\mathbf{W}) = -\sum_{k=1}^{3} m_k Log(s_k) \\ \mathbf{F}_{\eta}(\mathbf{W}) = -\sum_{k=1}^{3} m_k Log(s_k) \mathbf{U}_k \end{cases}$$
(4)

Main properties:

• The homogeneous convective subset (left hand side of (1)) is hyperbolic unless $U_1 \cdot n - U_k \cdot n = \pm c_k$, where n is a normal unit vector in \mathbb{R}^3 . Its eigenvalues are:

 $\lambda_{1,2,3,4,5}(\mathbf{W}) = U_1 \cdot n ; \ \lambda_{6,7,8}(\mathbf{W}) = U_2 \cdot n ; \ \lambda_{9,10,11}(\mathbf{W}) = U_3 \cdot n$ $\lambda_{12,13}(\mathbf{W}) = U_1 \cdot n \pm c_1; \lambda_{14,15}(\mathbf{W}) = U_2 \cdot n \pm c_2; \ \lambda_{16,17}(\mathbf{W}) = U_3 \cdot n \pm c_3$

• The entropy inequality for smooth solutions of (1) reads:

$$\frac{\partial \eta(\mathbf{W})}{\partial t} + \nabla \mathbf{F}_{\eta}(\mathbf{W}) \le 0.$$
(5)

• In the **n**-direction, system (1) admits unique jump conditions within each

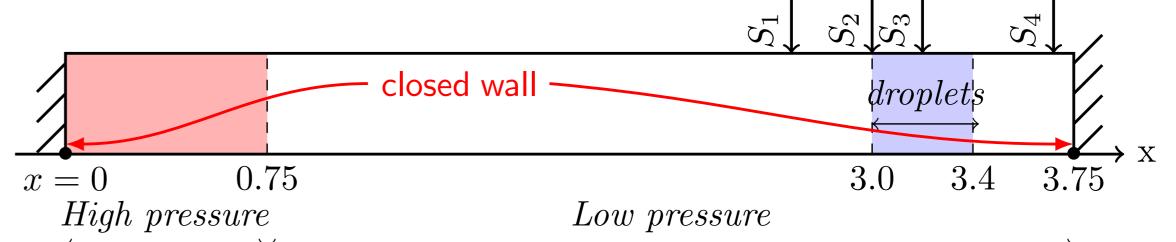
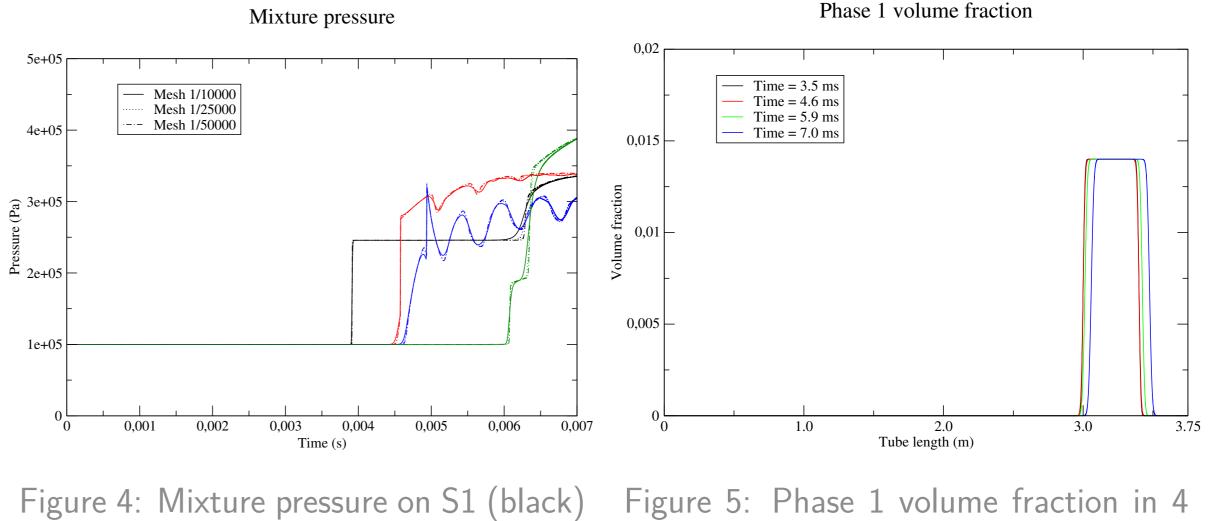


Figure 3: Sketch of the experimental shock tube apparatus.

Simulation results for solid droplet test case:



time instants.

Bibliography:

S2 (red) S3 (blue) S4 (green).

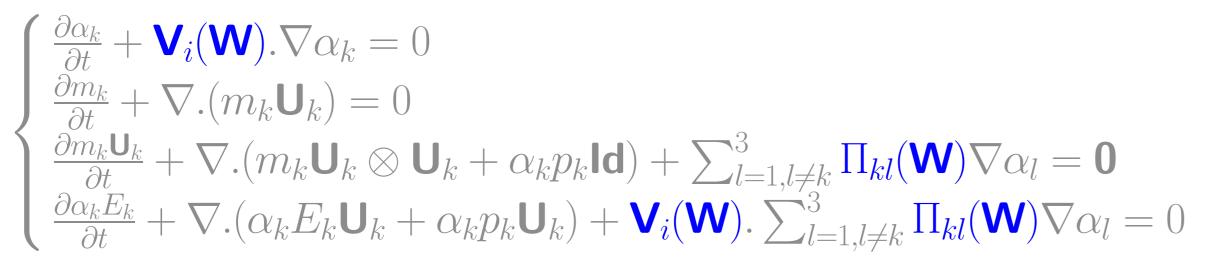
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NUMERICAL SCHEME

In the spirit of [2], a fractional step method is introduced to compute approximate solutions of (1) as follows:

• Step 1 - Evolution step : For a given initial condition W_i^n , compute an approximate solution of W at time t_{n+1} , namely $W^{n+1,-}$, by solving:



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Contacts: Hamza BOUKILI, hamza.boukili@edf.fr Jean-Marc HÉRARD, jean-marc.herard@edf.fr EDF Lab Chatou MFEE Department 6, quai Watier 78400 Chatou